IN THE DRAWINGS:

The attached replacement drawing sheets and annotated drawing sheet include a change to Figures 2 and 10. Figure 2 has been amended to add reference numbers to opposing radial bores of the ting –type joints. Figure 10 has been amended to include reference numeral 46. No new subject matter has been added.

Attachment: Replacement Sheets

Mark-up Drawing Sheet

IN THE SPECIFICATION:

A. Please replace the paragraph starting on page 13 line 15 with the following amended paragraph:

Figure 1 shows a perspective view of a constant velocity ~~first~~ torque transfer device;

B. Please insert the following paragraph immediately after the paragraph describing Figure 9, starting on line 5 of page 14:

Figure 10 shows a cross-sectional view another view of the constant velocity torque transfer device.

C. Please replace the paragraph starting on page 16, line 19 with the following amended paragraph:

For each ring-type joint 100, a first pair of opposing radial bores ~~12~~ 112 define a first pivot axis with the intermediate shaft 40, while a second pair of the bores ~~12~~ 112 defines a second pivot axis with one of the shafts 20, 30. As will be appreciated, once the intermediate shaft 40 and the first and second shafts 20, 30 are interconnected via the external rings 110, these components 20, 30 and 40 have two rotational degrees of freedom relative to one another, thereby allowing the two interconnected members to rotate about non-aligned axes of rotation. Furthermore, due to the geometry of the external rings 110, these components have a greater allowable range of motion as compared to such components interconnected with a cruciform type joint.

D. Please replace the paragraph starting on page 17, line 6 with the following amended paragraph:

As noted, the torque transfer device 10 also includes a centering mechanism 60 for maintaining the strict geometrical relationship between the first and second shafts 20, 30 required for constant velocity operation. As shown in Figs. 2-4, the centering mechanism 60 contains a cam tube 70 that is rotatably received within an internal bore 46 of the intermediate shaft 40. Disposed within the cam tube 70 is a linkage having first and second linkage members. ~~extending thertherough~~. Specifically, the first linkage member as shown in Figs. 2-4 and 10 is formed as a first connecting rod 80 extending between one end of the cam tube 70 and an internal bore 24 of the first shaft 20, and the second linkage member is formed as a second connecting rod 90 extending between the other end of the cam tube 70 and an internal bore 34 of the second drive shaft 30.

E. Please replace the paragraph starting on page 19 line 19 with the following amended paragraph:

Of note, the angle β between each sockets 72a, 72b of the cam tube 70 and the rotational axis of the intermediate shaft 40 and the angles β formed by the straight sections 82, 84 and 92, 94 of the connecting rods 80, 90 are equal. In this regard, when the centering mechanism 60 is assembled the connecting rods 80, 90 can rotate to a position such that the second bearing surfaces 88, 98 are disposed at a 0° angle relative to one another (see Figure 3). That is, each connecting rod 80, 90 may be rotated relative to reference line A-A (see Figure 3) such that the various angles cancel out allowing the first and second shafts 20, 30 to be aligned. Alternatively, the centering mechanism components 70, 80 and 90 can rotate such that the angles are additive. In this regard, the first and second shafts 20, 30 may be misaligned with respect to one another at an angle up to four times the angle β. See Figure 4. Furthermore, the centering mechanism components 70, 80 and 90 may be aligned such that the first and second shafts may be disposed at any angle between 0° and 4β. In the embodiment shown, β corresponds to a 10° angle such that the first and second shafts 20, 30 may be disposed relative to one another in a range of 40°. Though discussed primarily as being 10° the angle β may be varied between 1° and 20°. ~~However~~ Further, it will be appreciated that the invention is not limited to this angular operating range and other angular ranges are possible and within the scope of the present invention.